

Tue 7 Mar

736 127

Mixing entropy — more in tutorial

Pressure  $P = -\frac{\partial \langle E \rangle}{\partial V} \Big|_S$  connected to mechanical work

$$P = \frac{-F\Delta x}{-A\Delta x} = \frac{F}{A} = \frac{-W}{\Delta V}$$

$$\underline{W = \Delta \langle E \rangle = F\Delta x = -P\Delta V} \quad (\text{assumed constant entropy})$$

→  $\langle E \rangle$  can also change due to changing entropy

$$\Delta \langle E \rangle = W + \underline{\text{more}}$$

$W = -P\Delta V$  holds whether or not entropy changes

infinitesimal generalizes to  $W = - \int_{V_0}^{V_f} P dV$

Ideal gas law:  $P(V) = \frac{NkT}{V}$

$W > 0$  is work done on gas by surroundings (raises energy)

$W < 0$  is work done by gas, reduces  $\langle E \rangle$

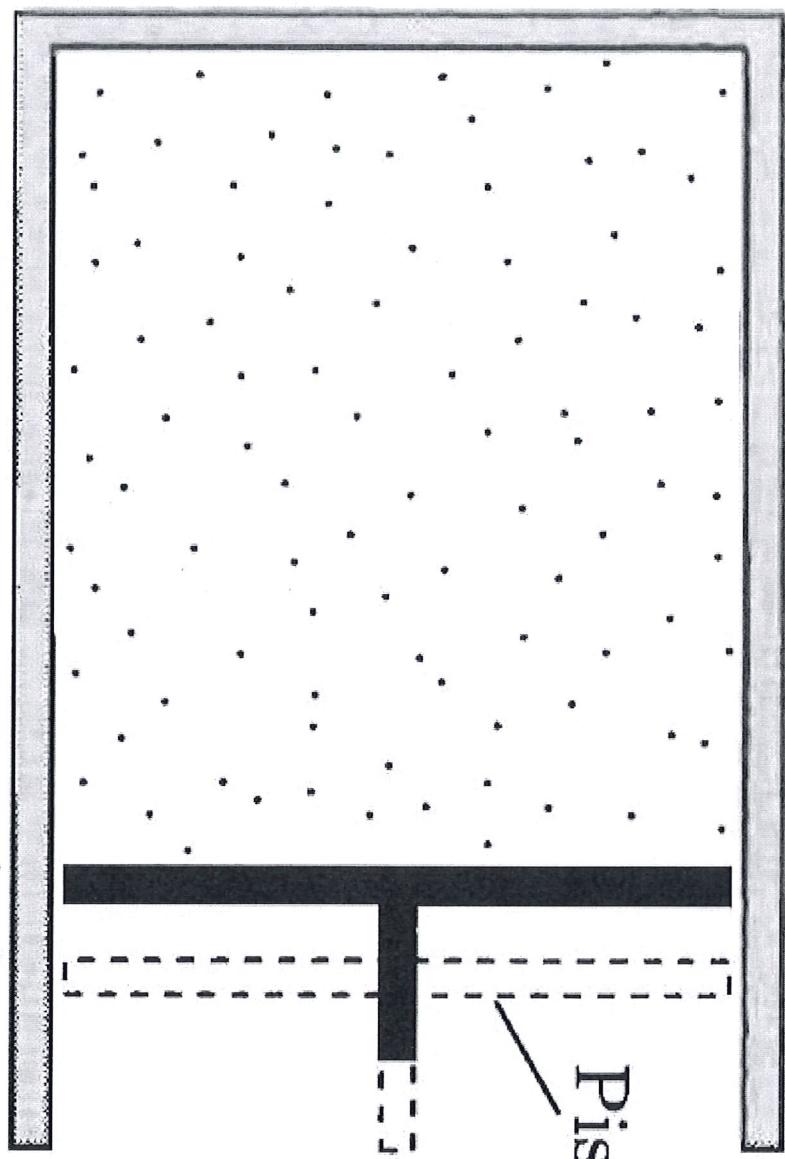
How can  $\langle E \rangle$  change without work?

Keep  $V$  constant  $\rightarrow W=0$

Change temperature  $\uparrow T$   
 $\langle E \rangle = \frac{3}{2} N k T \rightarrow \Delta \langle E \rangle = \frac{3}{2} N \underline{dT}$

Canonical ensemble demands fixed  $T$

We considering different canonical systems in equilibrium  
with different thermal reservoir  $\rightarrow T_i, T_f$



$$\Delta V = -A \Delta x$$

$$W = F \Delta x = A \langle E \rangle$$

$$d\langle E \rangle = \frac{3}{2} N \frac{dT}{T}$$

→ Expect change in entropy

$$S = N \log(\lambda_{th}^{-3}) + T\text{-indep.}$$

$$\lambda_{th} \propto \frac{1}{\sqrt{T}}$$

$$= N \log(T^{3/2}) + T\text{-indep.}$$

page 70

$$dS = d\left(\frac{3}{2} N \log T\right) = \frac{3}{2} N \frac{dT}{T} = \frac{d\langle E \rangle}{T}$$

$$\rightarrow d\langle E \rangle = T dS$$

This change in internal energy with constant  $V, N$

is the heat  $Q = T dS$

$$\rightarrow Q = \int_{S_1}^{S_2} T(S) dS$$

assuming  $S(T)$  invertible

With no work  $d\langle E \rangle = Q = \int T dS$

Energy and entropy not being created or destroyed

Instead (reversibly) flowing back & forth  
between system & surroundings

Isen<sub>trop</sub>ic processes  $dS = 0$

More generally, no heat transfer,  $Q = 0$

→ adiabatic process

Adiabatic + reversible = isentropic

In practice, adiabatic processes must be fast  
no time for heat exchange

Other extreme of slow processes → all possible heat exchange

Keeps temperature fixed to  $T$  of reservoir  
(isothermal)

Real processes in between extremes - usually closer to adiabatic

Putting things together

Invert  $S(T, V) \rightarrow T(S, V)$  fixing  $N$

$$\langle E \rangle(T, V) \rightarrow \langle E \rangle(S, V)$$

Taylor expand change in energy

$$\langle E \rangle(S, V) = \langle E \rangle(S_0, V_0) + (S - S_0) \left. \frac{\partial \langle E \rangle}{\partial S} \right|_V + (V - V_0) \left. \frac{\partial \langle E \rangle}{\partial V} \right|_S + \dots$$

$$d\langle E \rangle = dS(T) + dV(-P) = TdS - PdV = Q + W$$

Generalized first law

Any change in system's internal energy matched  
by heat exchange w/ surroundings  
or work done (or both)

Other key ideal gases equations

$$\langle E \rangle = \frac{3}{2} NT \quad S \propto VT^{3/2} \quad (\text{constant})$$

$$\text{EoS (ideal gas law)} \quad PV = NT$$

Sequence of processes that returns to initial macro-state  
 $(P, V, T, \langle E \rangle)$

~~Absent~~ ~~cyc~~

Thermodynamic cycle can be repeated  
to do work, transfer heat, etc.  
(engine) (refrigerator)

Fixing  $N \rightarrow$  all info contained in pressure and volume

$$T = \frac{PV}{N} \quad \langle E \rangle = \frac{3}{2} NT = \frac{3}{2} PV \quad S \propto V \left( \frac{PV}{N} \right)^{3/2}$$

Useful tool: Represent macro-state as a point  
in pressure-volume plane

## \*PV diagram

State ~ point

Process ~ line

Cycle ~ closed path

